

# Sliding Mode Observer-based Multi-fault Reconstruction of Nonlinear System

Changfan Zhang<sup>1</sup>, Houguang Chu<sup>1</sup>, Jing He<sup>\*1,2</sup>, Lin Jia<sup>3</sup>, Miaoying Zhang<sup>1</sup>

(1.College of Electrical and Information Engineering, Hunan University of Technology, Zhuzhou Hunan 412007; 2.College of Mechatronics and Automation National University of Defense Technology Changsha 410073; 3.College of Electrical and Information, Hunan University, Changsha, 410082, Hunan, China)

**Abstract:** For a class of nonlinear system with multi-dimensional actuator faults and multi-dimensional unknown input disturbances, the twice-linear coordinate transformation is made and the sliding mode observer is designed to obtain two dynamic deviation equations. One equation only deals with the faults and the other deals with both the faults and the disturbances. By means of the principle of equivalence of sliding mode observer and sliding mode variable structure, both the actuator faults and the unknown input disturbances can be reconstructed. The effectiveness of such method can be verified by the simulation example.

**Key Words:** coordinate transformation, robust fault diagnosis, fault reconstruction, sliding mode observer

## 1 INTRODUCTION

The robust fault diagnosis aims at detecting the minor fault as possible under the condition of insensitive unknown input disturbance such as model uncertainty. Relative to linear system, the nonlinear system is more complicated and the research is more difficult. However, the study of the nonlinear system with identified variables is practicable as some new control methods have been introduced in recent decades. For example, the sliding mode control [1] and adaptive control have been successfully applied in the fault detection and isolation. As the given phase locus is irrelevant to the external factors such as the parameters and disturbance of the controlled object, the system is robust to the unknown input disturbance when it moves on the sliding mode surface. Since the sliding mode theory was advanced to design the nonlinear system of observer[2], it had been successfully introduced in the model-based fault diagnosis.

The sliding mode observer was employed to maintain the sliding mode motion and obtain the fault reconstruction signal by analyzing the equivalent intervenient output signal, namely the equivalent control system occurred in the sliding model feedback control system[3]-[4]. Based on this method, a nonlinear system with uncertain variables was discussed in [5]. Such uncertain variables could be regarded as the actuator faults and unknown input disturbances. A sliding mode observer was designed to estimate the status and its intrinsic nature was used to reconstruct the faults. A coordinate transformation method was proposed and applied to the estimation of the uncertainties[6], this transformation was employed to launch a disturbance decoupling method for the weak and incipient actuator fault and carry out the fault detection by means of the residue signal generated by the Luenberger observer[7]. The nonlinear system in[8] comprised an actuator fault and an un-

known disturbance only. The equivalent principle of sliding mode variable structure and the coordinate transformation were used to reconstruct the actuator fault and estimate the unknown input disturbance. A nonlinear system with more faults was discussed in [9]. The system contained the faults only. The faults were reconstructed by the designed sliding mode observer and their status was estimated. The nonlinear systems in [10] and [11] contain either an actuator fault and an unknown disturbance or multi-dimensional faults only. The nonlinear systems in [12] and [13] comprised multi-dimensional faults and multi-dimensional disturbances. Two new systems were obtained by linear coordinate transformation. The faults are reconstructed by the designed adaptive observer and sliding mode observer but the disturbances are not estimated. Compare to the nonlinear system, the linear system is more simple. In [14] four observers were designed for a linear over-actuated system to perform the positioning and estimation of fault. In [15] and [16], a type of linear systems are discussed to carry out fault reconstruction by the designed observer. The above solution is limited in case of the occurrence of both multi-dimensional faults and multi-dimensional unknown input disturbances in nonlinear system. Based on Literatures [7] and [8], a fault diagnosis scheme of nonlinear system including a model with uncertain fault and unknown input disturbance is initiated in this paper. Providing the geometric conditions are satisfied, the nonlinear system model is converted to two subsystems via linear coordinate transformation twice, in which one is relevant to the faults only and the other is affected by both the actuator faults and unknown input disturbances. The sliding mode observers are constructed respectively for the two subsystems to obtain the dynamic deviation equations and reconstruct and estimate the multi-dimensional fault and multi-dimensional disturbance.

The rest of this paper is organized as follows. Section

\*Author for correspondence, E-mail: hejing@263.net.

2 describes the system and introduces some assumptions. Section 3 presents the coordinate transformation method and the design of sliding mode observer respectively. Section 4 proposes the method for reconstruction of faults and disturbances. In Sec. 5, computer simulation experiment is presented to demonstrate the effectiveness of the proposed strategy. Section 6 makes some concluding remarks.

## 2 Problem description

Consider an uncertain nonlinear system affected by actuator faults and unknown disturbances

$$\begin{cases} \dot{x}(t) = Ax + \Phi(x, u) + Ef_a(t) + Dd(t) + Bu(t) \\ y(t) = Cx(t) \end{cases} \quad (1)$$

where  $x \in R^n$ ,  $u \in R^m$  and  $y \in R^p$  denote respectively the state variables, inputs and outputs;  $\Phi(x, u) \in R^n$  is a known nonlinear function.  $d(t) \in R^q$  is an unknown nonlinear function lumped uncertainties and disturbances experienced by the system which is assumed to be bounded, i.e. there exists a positive constant  $\gamma_1$  such that  $\|d(t)\| \leq \gamma_1$ . The unknown nonlinear function  $f_a(t) \in R^q$  represents actuator faults, which supposed to be norm bounded, i.e. there exists a constant  $\gamma_2$  such that  $\|f_a(t)\| \leq \gamma_2$ .  $A \in R^{n \times n}$ ,  $B \in R^{n \times m}$ ,  $C \in R^{p \times n}$ ,  $D \in R^{n \times q}$  and  $E \in R^{n \times q}$  are known constant matrices, where  $D$  is the known distribution matrix of disturbances,  $E$  is the known constant distribution matrix of actuator faults and  $n > p > q$ .

Assumption 1.  $D$  is a column full rank matrix, and  $\text{rank}(CD) = \text{rank}(D)$ .

Assumption 2. The matrix pair  $(A, C)$  is observable.

There exist the two transform matrixes  $T$  and  $S$  [6] such that:

$$\begin{aligned} x(t) &= T^{-1}z(t) = T^{-1} \begin{bmatrix} z_1(t) \\ z_2(t) \end{bmatrix} & y(t) &= S^{-1} \begin{bmatrix} v_1(t) \\ v_2(t) \end{bmatrix} \end{aligned}$$

and system (1) can be transformed as:

$$\begin{cases} \dot{z}(t) = \begin{bmatrix} \dot{z}_1(t) \\ \dot{z}_2(t) \end{bmatrix} = TAT^{-1}z(t) \\ \quad + T\Phi(z, u) + TEf_a(t) \\ \quad + TDD(t) + TBU(t) \\ v(t) = \begin{bmatrix} v_1(t) \\ v_2(t) \end{bmatrix} = SCT^{-1}z(t) \end{cases} \quad (2)$$

where  $SCT^{-1} = \begin{bmatrix} C_{11} & 0 \\ 0 & C_{22} \end{bmatrix}$ , and  $C_{22}$  is an invertible matrix.

Selected a nonsingular transformation matrix  $T$  [6]:

$$T = \begin{bmatrix} I_{n-q} & -D_1D_2^{-1} \\ 0 & I_q \end{bmatrix}$$

and each matrix in (2) is:

$$\begin{aligned} TAT^{-1} &= \begin{bmatrix} \bar{A}_{11} & \bar{A}_{12} \\ \bar{A}_{21} & \bar{A}_{22} \end{bmatrix} \\ TD &= \begin{bmatrix} 0 \\ \bar{D}_2 \end{bmatrix}, TB = \begin{bmatrix} \bar{B}_1 \\ \bar{B}_2 \end{bmatrix} \end{aligned}$$

$$TE = \begin{bmatrix} \bar{E}_1 \\ \bar{E}_2 \end{bmatrix}, T\Phi(x, u) = \begin{bmatrix} \bar{\Phi}_1(x, u) \\ \bar{\Phi}_2(x, u) \end{bmatrix}$$

System (1) can be transformed into the following two subsystems:

$$\begin{cases} \dot{z}_1(t) = \bar{A}_{11}z_1(t) + \bar{A}_{12}z_2(t) + \bar{\Phi}_1(z, u) \\ \quad + \bar{E}_1f_a(t) + \bar{B}_1u(t) \\ v_1(t) = C_{11}z_1(t) \end{cases} \quad (3)$$

$$\begin{cases} \dot{z}_2(t) = \bar{A}_{21}z_1(t) + \bar{A}_{22}z_2(t) + \bar{\Phi}_2(z, u) \\ \quad + \bar{E}_2f_a(t) + \bar{D}_2d(t) + \bar{B}_2u(t) \\ v_2(t) = C_{22}z_2(t) \end{cases} \quad (4)$$

## 3 Sliding mode observer design

Assumption 3.  $(\bar{A}_{11}, C_{11})$  and  $(\bar{A}_{22}, C_{22})$  are observable[7].

Assumption 4. The nonlinear functions  $\bar{\Phi}_1$  and  $\bar{\Phi}_2$  are assumed to be known and satisfy the Lipschitz conditions such that

$$\|\bar{\Phi}_1(z, u) - \bar{\Phi}_1(\hat{z}, u)\| \leq \gamma_3 \|z - \hat{z}\| \leq \gamma_3(\|e_1\| + \|e_2\|)$$

$$\|\bar{\Phi}_2(z, u) - \bar{\Phi}_2(\hat{z}, u)\| \leq \gamma_4 \|z - \hat{z}\| \leq \gamma_4(\|e_1\| + \|e_2\|)$$

where  $\gamma_3$  and  $\gamma_4$  are known positive constants called Lipschitz constant.

Assumption 5.  $F_1$  and  $F_2$  satisfy the following equations

$$P_1\bar{E}_1 = C_{11}^T F_1^T$$

$$P_2\bar{E}_2 = C_{22}^T F_2^T$$

where  $P_1$  and  $P_2$  are symmetric positive definite matrices.

From Assumption 3 we know that there exist matrices  $L_1$  and  $L_2$  which make  $A_{01}$  and  $A_{02}$  stable matrices:

$$\bar{A}_{11} - L_1\bar{C}_{11} = A_{10}$$

$$\bar{A}_{22} - L_2\bar{C}_{22} = A_{20}$$

Based on the transformed systems (3)(4), the present study proposes the following two sliding mode observers described as:

$$\begin{cases} \dot{\hat{z}}_1(t) = \bar{A}_{11}\hat{z}_1(t) + \bar{A}_{12}\hat{z}_2(t) + \bar{\Phi}_1(\hat{z}, u) \\ \quad + \bar{E}_1r_1(t) + \bar{B}_1u(t) + L_1(v_1(t) - \hat{v}_1(t)) \\ \hat{v}_1(t) = C_{11}\hat{z}_1(t) \end{cases} \quad (5)$$

$$\begin{cases} \dot{\hat{z}}_2(t) = \bar{A}_{21}\hat{z}_1(t) + \bar{A}_{22}\hat{z}_2(t) + \bar{\Phi}_2(\hat{z}, u) \\ \quad + \bar{E}_2r_2(t) + \bar{B}_2u(t) + L_2(v_2(t) - \hat{v}_2(t)) \\ \hat{v}_2(t) = C_{22}\hat{z}_2(t) \end{cases} \quad (6)$$

where  $\hat{z}_1$ ,  $\hat{z}_2$  and  $\hat{v}_1(t)$ ,  $\hat{v}_2(t)$  denote respectively the estimated states and outputs, and  $r_1(t)$ ,  $r_2(t)$  represents the input signals of sliding mode observers:

$$\begin{cases} r_1(t) = \rho_1 \text{sign}(F_1(C_{11}z_1 - C_{11}\hat{z}_1)) \\ r_2(t) = \rho_2 \text{sign}(F_2(C_{22}z_2 - C_{22}\hat{z}_2)) \end{cases} \quad (7)$$

where  $\rho_1$  and  $\rho_2$  are all to be designed. By defining the state estimation errors  $e_1(t) = z_1(t) - \hat{z}_1(t)$ ,  $e_2(t) = z_2(t) - \hat{z}_2(t)$  and output estimation errors  $e_{v1}(t) = v_1(t) - \hat{v}_1(t) = C_{11}e_1(t)$ ,  $e_{v2}(t) = v_2(t) - \hat{v}_2(t) = C_{22}e_2(t)$ . we order  $e = (e_1 \ e_2)^T$ . The observation-error dynamic equation can be computed from (3) to (6) as:

$$\dot{e}_1(t) = (\bar{A}_{11} - L_1 C_{11}) e_1(t) + \bar{A}_{12} e_2(t) + \bar{\Phi}_1(z, u) - \bar{\Phi}_1(\hat{z}, u) + \bar{E}_1 f_a(t) - \bar{E}_1 r_1(t) \quad (8)$$

$$\dot{e}_2(t) = \bar{A}_{21} e_1(t) + (\bar{A}_{22} - L_2 C_{22}) e_2(t) + \bar{\Phi}_2(z, u) - \bar{\Phi}_2(\hat{z}, u) + \bar{E}_2 f_a(t) - \bar{E}_2 r_2(t) + \bar{D}_2 d(t) \quad (9)$$

Lemma 1 Consider the error dynamics system (8), (9), assumption 4-5, and if the following LMI is satisfied

$$\begin{bmatrix} \bar{A}^T P + P \bar{A} + \xi \gamma^2 I_n & P \\ P & -\xi I_n \end{bmatrix} < 0 \quad (10)$$

the parameters  $\rho_1$  and  $\rho_2$  satisfy

$$\rho_1 > \gamma_2$$

$$\rho_2 > \gamma_2 + \frac{\|\bar{D}_2\|}{\|\bar{E}_2\|} \gamma_1$$

then  $e_1$  and  $e_2$  are asymptotically convergent, that is

$$\lim_{t \rightarrow \infty} e_1(t) = 0, \quad \lim_{t \rightarrow \infty} e_2(t) = 0$$

where  $\xi$  is a positive constant and  $I_q$  is a q-dimensional identity matrix.

Proof. Consider the following Lyapunov function

$$V = e_1^T P_1 e_1 + e_2^T P_2 e_2 \quad (11)$$

Let

$$\bar{A} = \begin{bmatrix} \bar{A}_{11} - L_1 C_{11} & \bar{A}_{12} \\ \bar{A}_{21} & \bar{A}_{22} - L_2 C_{22} \end{bmatrix}$$

$$P = \begin{bmatrix} P_1 & 0 \\ 0 & P_2 \end{bmatrix}$$

The derivative of  $V$  along with the error dynamic systems (8) and (9) is

$$\begin{aligned} \dot{V} &= e^T (\bar{A}^T P + P \bar{A}) e + 2e^T P (\bar{\Phi}(z, u) - \bar{\Phi}(\hat{z}, u)) \\ &+ 2e_1^T P_1 (\bar{E}_1 f_a(t) - \bar{E}_1 r_1(t)) \\ &+ 2e_2^T P_2 (\bar{E}_2 f_a(t) + \bar{D}_2 d(t) - \bar{E}_2 r_2(t)) \end{aligned} \quad (12)$$

we obtain

$$\dot{V} \leq e^T (\bar{A}^T P + P \bar{A} + \frac{1}{\xi} P^2 + \xi \gamma^2 I_n) e \quad (13)$$

$\dot{V}$  turns out to be negative definite by imposing

$$\bar{A}^T P + P \bar{A} + \frac{1}{\xi} P^2 + \xi \gamma^2 I_n < 0 \quad (14)$$

The linear matrix inequality is satisfied

$$\begin{bmatrix} \bar{A}^T P + P \bar{A} + \xi \gamma^2 I_n & P \\ P & -\xi I_n \end{bmatrix} < 0 \quad (15)$$

This completes the proof.

Thus  $\dot{V}_1 < 0$  as long as  $e(t) \neq 0$ , so that  $e(t) = 0$  is a globally asymptotically stable equilibrium point. That is

$$\lim_{t \rightarrow \infty} e_1(t) = 0 \quad \lim_{t \rightarrow \infty} e_2(t) = 0 \quad (16)$$

Lemma 2 Choose the sliding mode surfaces  $s_1 = F_1 e_{v1}$ ,  $s_2 = F_2 e_{v2}$ , based on Assumptions 4 and 5, then the error system (8)(9) will be driven to the sliding mode surface in finite time.

Proof. (1) Selecting the Lyapunov function as follows:

$$V_1 = \frac{1}{2} s_1^T s_1 \quad (17)$$

The time derivative of  $V_1$  along the trajectories of system (8) is given by

$$\begin{aligned} \dot{V}_1 &= (F_1 C_{11} e_1)^T (F_1 C_{11} \dot{e}_1) = \\ &(F_1 C_{11} e_1)^T F_1 C_{11} ((\bar{A}_{11} - L_1 C_{11}) e_1(t) + \bar{A}_{12} e_2(t) \\ &+ \bar{\Phi}_1(z, u) - \bar{\Phi}_1(\hat{z}, u) + \bar{E}_1 f_a(t) - \bar{E}_1 r_1(t)) \end{aligned} \quad (18)$$

We design  $\rho_1$  which satisfies

$$\rho_1 \geq \frac{(\|\bar{A}_{11} - L_1 C_{11}\| + \gamma_4) \delta_1 + (\|\bar{A}_{12}\| + \gamma_3) \delta_2 + K_1}{\|\bar{E}_1\|} + \gamma_2 \quad (19)$$

then

$$\dot{V}_1 \leq -K_1 \|F_1 C_{11}\| \|s_1\| \quad (20)$$

where  $K_1$  is a positive constant.

This shows that the reachability condition [17] is satisfied. As a consequence, an ideal sliding motion will take place on the surface  $s_i = 0$  ( $i = 1, 2$ ) and after some finite time we have

$$F_1 e_{v1} = F_1 \dot{e}_{v1} = F_1 C_{11} \dot{e}_1 = 0 \quad (21)$$

$$F_2 e_{v2} = F_2 \dot{e}_{v2} = F_2 C_{22} \dot{e}_2 = 0 \quad (22)$$

#### 4 Fault reconstruction and disturbance Estimation

From lemma 1,  $e_1$  and  $e_2$  will approach zero in some finite time and  $\Phi(z, u) = \bar{\Phi}(\hat{z}, u)$ . lemma 2 show that the system reaches the sliding mode surface,  $s_i = \dot{s}_i = 0$  ( $i = 1, 2$ ) according to the sliding mode equivalent principle [17]. With these conclusions, we can reconstruction the actuator faults and estimate the unknown input disturbances

(1) The reconstruction of actuator faults

From (21), we have  $F_1 C_{11} \dot{e}_1 = 0$ , then

$$\begin{aligned} F_1 C_{11} ((\bar{A}_{11} - L_1 C_{11}) e_1(t) + \bar{A}_{12} e_2(t) + \bar{\Phi}_1(z, u) \\ - \bar{\Phi}_1(\hat{z}, u) + \bar{E}_1 f_a(t) - \bar{E}_1 r_1(t)) = 0 \end{aligned} \quad (23)$$

With lemma 1 and (7),we have

$$f_a(t) \approx r_1(t) = \rho_1 \text{sgn}(F_1 e_{v1}) \quad (24)$$

(2)The estimation of disturbances

From (22),we have  $F_2 C_{22} \dot{e}_2 = 0$ , then

$$\bar{A}_{21} e_1(t) + (\bar{A}_{22} - L_2 C_{22}) e_2(t) + \bar{\Phi}_2(z, u) - \bar{\Phi}_2(\hat{z}, u) + \bar{E}_2 f_a(t) - \bar{E}_2 r_2(t) + \bar{D}_2 d(t) = 0 \quad (25)$$

With lemma 1 and (7),we have

$$\begin{aligned} d(t) &\approx \bar{D}_2^{-1} (\bar{E}_2 r_2(t) - \bar{E}_2 f_a(t)) \\ &= \bar{D}_2^{-1} \bar{E}_2 (\rho_2 \text{sgn}(F_2 e_{v2}) - \rho_1 \text{sgn}(F_1 e_{v1})) \end{aligned} \quad (26)$$

To weaken the chattering we define

$$\text{sign}(e_y) = \frac{e_y}{|e_y| + \sigma_1} \quad \text{sign}(e_{v2}) = \frac{e_{v2}}{|e_{v2}| + \sigma_2}$$

where  $\sigma_1$  and  $\sigma_2$  are two small positive constant.

## 5 Simulation result

Consider a single-link robotic arm with a revolute elastic joint rotating in a vertical plane whose motion equations are [18]

$$\begin{cases} J_1 \ddot{q}_1 + F_l \dot{q}_1 + k(q_1 - q_2) + mgl \sin q_1 = 0 \\ J_m \ddot{q}_2 + F_m \dot{q}_2 - k(q_1 - q_2) = u \end{cases} \quad (27)$$

where  $q_1$  and  $q_2$  are the link displacement and the rotor displacement, respectively. The link inertia  $J_1$ , the motor rotor inertia  $J_m$ , the elastic constant  $k$ , the link mass  $m$ , the gravity  $g$ , the connecting rod length  $l$ , and the viscous friction coefficients  $F$ ,  $F_m$  are all positive constant parameters. The control  $u$  is the torque delivered by the motor.

We define the state variables  $x_1 = q_1$ ,  $x_2 = \dot{q}_1$ ,  $x_3 = q_2$  and  $x_4 = \dot{q}_2$ . It is known in practice the parameters of robot vary as time changes with handling different objects. Such changing parameters are defined as unknown input disturbances and actuator fault. Assuming there are two actuator faults and two unknown input disturbances on the single-articulation robot, the system equation as shown Equation (1) can be obtained.

Correspond (1), the parameter matrixes are

$$A = \begin{bmatrix} 0 & 1 & 0 & 0 \\ \frac{-k}{J_1} & \frac{-F_l}{J_1} & \frac{k}{J_1} & 0 \\ 0 & 0 & 0 & 1 \\ \frac{k}{J_m} & 0 & \frac{-k}{J_m} & \frac{-F_m}{J_m} \end{bmatrix}$$

$$f(x, u, t) = \begin{bmatrix} 0 \\ -0.2205 \sin x_1 \\ 0 \\ 0 \end{bmatrix}$$

$$E = \begin{bmatrix} 1 & 0 \\ 0 & 2 \\ 2 & 0 \\ 0 & 1 \end{bmatrix} \quad D = \begin{bmatrix} 1 & 0 \\ 0 & 0 \\ 1 & 0 \\ 0 & 2 \end{bmatrix} = \begin{bmatrix} D_1 \\ D_2 \end{bmatrix}$$

$$B = \begin{bmatrix} 1 \\ 0 \\ 0 \\ 1 \end{bmatrix} \quad C = \begin{bmatrix} 1 & 0 & 0 & 0 \\ 0 & 1 & 0 & 0 \\ 0 & 0 & 1 & 0 \\ 0 & 0 & 0 & 1 \end{bmatrix}$$

$$f_a(t) = \begin{bmatrix} f_1(t) \\ f_2(t) \end{bmatrix} \quad d(t) = \begin{bmatrix} d_1(t) \\ d_2(t) \end{bmatrix}$$

The transformational matrix  $T$

$$T = \begin{bmatrix} 1 & 0 & -1 & 0 \\ 0 & 1 & 0 & 0 \\ 0 & 0 & 1 & 0 \\ 0 & 0 & 0 & 1 \end{bmatrix}$$

Then

$$TE = \begin{bmatrix} -1 & 0 \\ 0 & 2 \\ 2 & 0 \\ 0 & 1 \end{bmatrix} = \begin{bmatrix} E_1 \\ E_2 \end{bmatrix}$$

$$TD = \begin{bmatrix} 0 & 0 \\ 0 & 0 \\ 1 & 0 \\ 0 & 2 \end{bmatrix} = \begin{bmatrix} 0 \\ D_2 \end{bmatrix}$$

So we obtain

$$D_2^{-1} = \begin{bmatrix} 1 & 0 \\ 0 & 0.5 \end{bmatrix}$$

The transformational matrix  $S$  is

$$S = \begin{bmatrix} 1 & 0 & 1 & 0 \\ 0 & 1 & 0 & 0 \\ 0 & 0 & 1 & 0 \\ 0 & 0 & 0 & 1 \end{bmatrix}$$

Design the matrices as  $P_1 = \begin{bmatrix} 2 & 0 \\ 0 & 1 \end{bmatrix}$ ,  $P_2 =$

$$\begin{bmatrix} 1 & 0 \\ 0 & 1 \end{bmatrix}, F_1 = \begin{bmatrix} -2 & 0 \\ 0 & 2 \end{bmatrix}, F_2 = \begin{bmatrix} 2 & 0 \\ 0 & 1 \end{bmatrix}, L_1 = \begin{bmatrix} 3 & -1 \\ 2 & 3.25 \end{bmatrix} \text{ and } L_2 = \begin{bmatrix} 4 & -1 \\ 2 & -1 \end{bmatrix}.$$

The actuator fault reconstruction algorithm

$$\hat{f}_a(t) \approx \rho_1 \text{sign}(F_1 e_{v1}) = \begin{bmatrix} \rho_{11} \frac{-2e_{11}}{|-2e_{11}| + \sigma_1} \\ \rho_{12} \frac{2e_{12}}{|2e_{12}| + \sigma_1} \end{bmatrix}$$

and the unknown disturbances

$$\begin{aligned} d(t) &\approx \bar{D}_2^{-1} \bar{E}_2 (\rho_2 \text{sgn}(F_2 e_{v2}) - \rho_1 \text{sgn}(F_1 e_{v1})) \\ &= \begin{bmatrix} 2 & 0 \\ 0 & 0.5 \end{bmatrix} \begin{bmatrix} \rho_{21} \frac{2e_{21}}{|2e_{21}| + \sigma_2} - \rho_{11} \frac{-2e_{11}}{|-2e_{11}| + \sigma_1} \\ \rho_{22} \frac{e_{22}}{|e_{22}| + \sigma_2} - \rho_{12} \frac{2e_{12}}{|2e_{12}| + \sigma_1} \end{bmatrix} \end{aligned}$$

The robot parameters are  $k = 2Nm/rad$ ,  $F_m = 1$ ,  $F_1 = 0.5Nm/(rad/s)$ ,  $J_m = 1Nm^2$ ,  $J_1 = 2Nm^2$ ,  $m = 0.15kg$ ,  $g = 9.8m/s^2$  and  $l = 0.3m$ . The input  $u = \sin(5t) + 4\sin(20t)$ . The designed parameters  $\rho_1 = [\rho_{11} \quad \rho_{12}]^T$ ,  $\rho_2 = [\rho_{21} \quad \rho_{22}]^T$ , where  $\rho_{11} = \rho_{12} = 40$ ,  $\rho_{21} = 80$  and  $\rho_{22} = 40$ , and  $\sigma = [\sigma_1 \quad \sigma_2]^T$ , where  $\sigma_1 = \sigma_2 = 0.01$ .  $e_{v1} = [e_{11} \quad e_{12}]^T$ ,  $e_{v2} = [e_{21} \quad e_{22}]^T$ . The initial value of state variable  $x$  in the simulation example are chosen as -2, -3, -3 and -2. Two

overlapped sine signals are used to simulate the incipient fault  $f_1$ , where  $f_1 = 2\sin 40t + 2\sin 5t$ . A 0.5s-cycle square wave is used as the given data of fault  $f_2$ , which is the combination of abrupt fault and intermittent fault. A white noise with amplitude of 3 is used as the unknown input disturbance  $d_1$ . The unknown input disturbance  $d_2$  is a sine wave, where  $d_2 = \sin 5t$ . The simulation results are as follows:

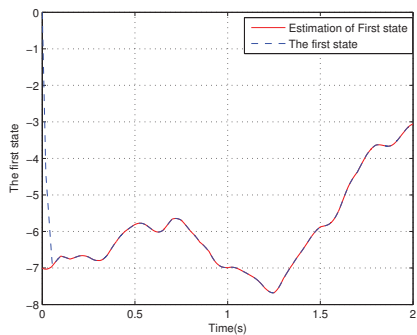


Figure 1: The first state  $x_1$  and its estimation  $\hat{x}_1$

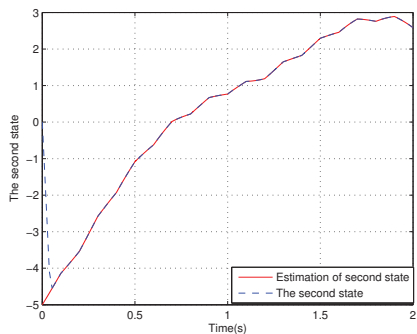


Figure 2: The second state  $x_2$  and its estimation  $\hat{x}_2$

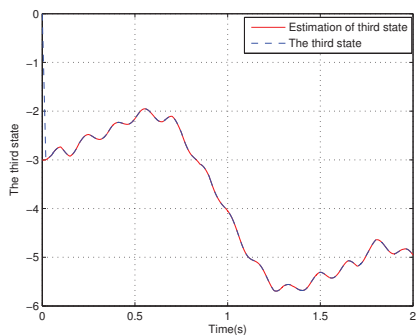


Figure 3: The third state  $x_3$  and its estimation  $\hat{x}_3$

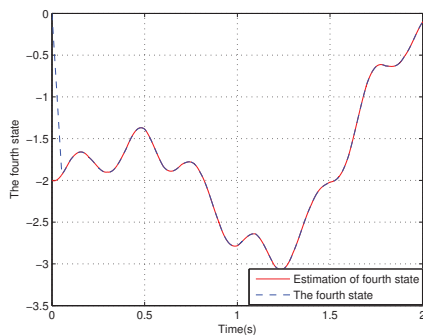


Figure 4: The fourth state  $x_4$  and its estimation  $\hat{x}_4$

Figures 1, 2, 3 and 4 respectively indicate for the estimations of statuses  $x_1$ ,  $x_2$ ,  $x_3$  and  $x_4$  reach the given data rapidly at the different initial data and remain stable. The tracking performance is excellent. The above simulation results imply that convergence rate of observer is high, which sets a foundation for the reconstruction of the multi-dimensional fault and unknown input disturbance in the context below.

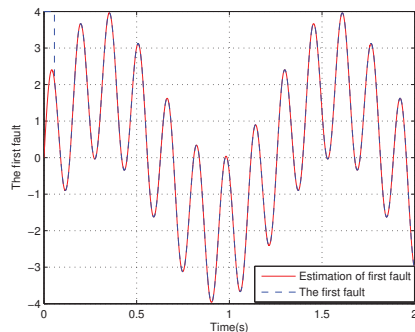


Figure 5: The fault  $f_1$  and its estimation  $\hat{f}_1$

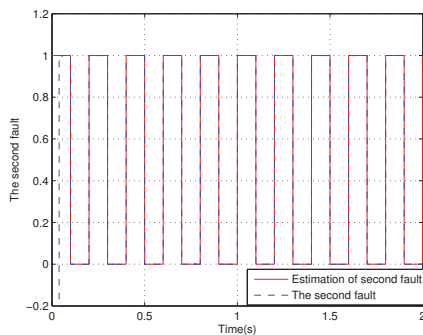


Figure 6: The fault  $f_2$  and its estimation  $\hat{f}_2$

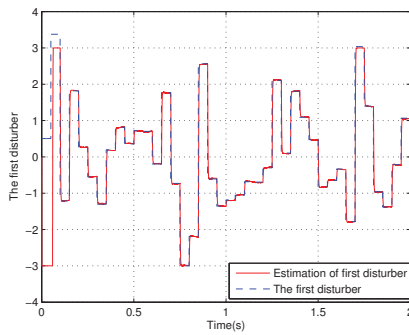


Figure 7: The disturbance  $d_1$  and its estimation  $\hat{d}_1$

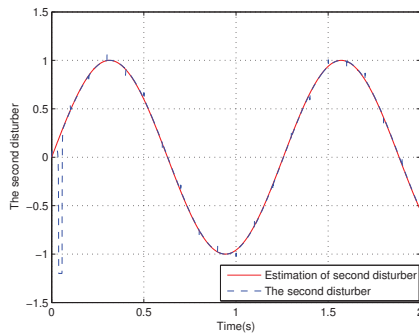


Figure 8: The disturbance  $d_2$  and its estimation  $\hat{d}_2$

Figures 5 respectively the reconstruction results of fault  $f_1$  and Figures 6 indicate the reconstruction of Fault  $f_2$ . The above simulation results imply both incipient fault  $f_1$  and abrupt fault  $f_2$  can be precisely reconstructed. Figures 7 and 8 indicate the reconstruction of unknown input disturbances. The tracking performance is well.

## 6 Conclusions

In this paper, a nonlinear system with multi-fault and multi-disturbance is studied. The coordinate transformation is made for the original system twice, in which one subsystem deals with only fault and the other deals with both fault and disturbance. The precise reconstruction of fault and disturbance is obtained by means of the sliding mode variable structure equivalent principle. The nonlinear system in the simulation example comprises 2-dimensional fault and 2-dimensional disturbance, each being precisely reconstructed. The simulation results verify the effectiveness of such method.

## 7 Acknowledgment

This study is supported by the Natural Science Foundation of China (61273157 and 61473117), Hunan Provincial Natural Science Foundation of China(No.14JJ5024), Hunan Province Education Department (No.12AO40 and No.13CY018).

## REFERENCES

[1] C. Edwards, S. K. Spurgeon, and R. J. Patton, Sliding mode observers for fault detection and isolation, *Automatica*, Vol.36, 541-553, 2000.

[2] B. Jiang, M. Staroswiecki, and V. Cocquempot, Fault estimation in nonlinear uncertain systems using robust/sliding-mode observers, *IEEE Proceedings - Control Theory and Applications*, Vol.151, 29C37, 2004.

[3] X. G. Yan and C. Edwards, Nonlinear robust fault reconstruction and estimation using a sliding mode observer, *Automatica*, Vol.43, 1605-1614, 2007.

[4] K. Y. Ng, C. P. Tan, C. Edwards, and Y. C. Kuang, New results in robust actuator fault reconstruction for linear uncertain systems using sliding mode observers, *Int. J. Robust Nonlinear Control*, Vol.17, 1294C1319, 2007.

[5] X. G. Yan and C. Edwards, Adaptive sliding-mode-observer-based fault reconstruction for nonlinear systems with parametric uncertainties, *IEEE Transactions On Industrial Electronics*, Vol.55, 2008.

[6] M. Corless and J. Tu, State and input estimation for a class of uncertain systems, *Automatica*, Vol.34, 757-764, 1988.

[7] W. Chen and F. N. Chowdhury, Design of sliding mode observers with sensitivity to incipient faults, *16th IEEE International Conference on Control Applications Part of IEEE International Conference on Automation and Logistics*, Vol.10, 795-800, 2007.

[8] J. He and C. F. Zhang, Fault reconstruction base on sliding mode observer for nonlinear systems, *Hindawi Mathematical Problems in Engineering*, Volume 2012.

[9] K. C. Veluvolu and Y.C. Soh, Nonlinear sliding mode observers for fault reconstruction and state estimation, *11th Int.Conf.Control, Automation, Robotics and Vision*, Singapore, 2010.

[10] N. Orani, A. Pisano and E. Usai, Exact reconstruction of actuator faults by reduced order sliding mode observer, *18th IEEE International Conference on Control Applications Part of 2009 IEEE Multi-conference on Systems and Control Saint Petersburg, Russia*, 2009.

[11] S. Dhahri, F. B. Hmida, A.Sellami and M.Gossa, Actuator fault reconstruction for nonlinear uncertain systems using sliding mode observer, *International Conference on Signals, Circuits and Systems*, 2009.

[12] J. X. Liu, S. Laghrouche, and M. Wack, Adaptive higher order sliding mode observer-based fault reconstruction for a class of nonlinear uncertain systems, *52nd IEEE Conference on Decision and Control, Florence*, 2013.

[13] J. Zhang, A. K. Swain and S.K. Nguang, Reconstruction of actuator fault for a class of nonlinear systems using sliding mode observer, *American Control Conference on O'Farrell Street, San Francisco, June 29-July 01, 2011*.

[14] Z. X. Yin, P. Yang, B. Jiang, Hua Lianghao, A design approach of multi-observers-based multi-faults estimation for over-actuate systems, *Proceeding of the 31st Chinese Control Conference, Hefei*, 2012.

[15] H. Y. Liu, Z. S. Duan, Actuator fault estimation using direct reconstruction approach for linear multivariable systems, *I-ET Control Theory Appl*, Vol.6, 141-148, 2012.

[16] K. Y. Ng, C. P. Tan and D. Oetomo, Enhance fault reconstruction using cascaded sliding mode observers, *12th IEEE Workshop on Variable Structure Systems, VSS'12, January 12-14, Mumbai*, 2012.

[17] F. Y. Wang, *Sliding Mode Variable Structure Control*. Beijing: Tsinghua University Press, 2005.

[18] X. Zhang, T. Parisini and M. M. Polycarpou, Sensor bias fault isolation in a class of nonlinear systems, *IEEE Transactions on Automatic Control*, Vol.50, 370-376, 2005.